



Research Article

Analysis of Chemical Reaction Kinetics Behavior of Nitrogen Oxide During Air-staged Combustion in Pulverized Boiler

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Abstract

Because the air-staged combustion technology is one of the key technologies with low investment running costs and high emission reduction efficiency for the pulverized boiler, it is important to reveal the chemical reaction kinetics mechanism for developing various technologies of nitrogen oxide reduction emissions. At the present work, a three-dimensional mesh model of the large-scale four corner tangentially fired boiler furnace is established with the GAMBIT pre-processing of the FLUENT software. The partial turbulent premixed and diffusion flame was simulated for the air-staged combustion processing. Parameters distributions for the air-staged and no the air-staged were obtained, including in-furnace flow field, temperature field and nitrogen oxide concentration field. The results show that the air-staged has more regular velocity field, higher velocity of flue gas, higher turbulence intensity and more uniform temperature of flue gas. In addition, a lower negative pressure zone and lower O₂ concentration zone is formed in the main combustion zone, which is conducive to the NO of fuel type reduced to N₂, enhanced the effect of NO_x reduction. Copyright © 2016 BCREC GROUP. All rights reserved

Keywords: Combustion; Pulverized coal; Air-staged; Chemical reaction kinetics mechanism of NO_x

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1. Introduction

In recent years, with the rapid development of national economy and electric power industry, the large-scale coal-fired power plants have been increasing constantly. The 300 MW and 600 MW units have been applied widely as the primary units in China. Both their high efficiency and low emission of Nitrogen Oxide (NO_x) play an important role in saving energy

and environmental protection. One of the most effective ways is to adopt low NO_x combustion technologies, such as Low oxygen combustion and Air-staged combustion. In the late 80s, a series of high capacities units with ABB-CE technology were introduced into China [1]. Low NO_x burner reduces NO_x by regulating velocity of both primary air and secondary air and mixture time of pulverized coal so as to form rich-fuel and lean-oxygen. This technology has been applied in utility boilers [2]. Air-staged combustion processing is one of the technologies that are applied widely both at home and aboard. Its principle is that 80-85% of required air is

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firstly supplied to the primary burner so as to decrease the Oxygen supply, which reduces the thermal NO and intermediate products. Moreover, the generated NO is reduced into N₂ in the lean-Oxygen atmosphere.

Hao [3] investigated experimentally the air-staged combustion of both the Shenmue bituminous coal and the Yangquan meager coal in a one-dimensional furnace. Both locations of staged-air and the burner pitch depends on the demand of oxygen at various periods of combustion. Wang *et al.* [4] analyzed numerically the effects of deep air-staged on the combustion of low volatile coal in the one-dimensional furnace. It is found that the fine pulverized coal shows the stronger reactivity and more oxygen consumption, reducing the NO_x formation. Therefore, NO_x emission in deep air-staged is dominated by the fineness of pulverized coal. This technology is more suitable for the higher-capacity units over 600 MW, in which the over-fire air accounts for 30% of overall air. However, if this technology was applied to those units less than 300 MW, the ratio of over-fire air to overall air needs to adjust to 15% [5]. Sun *et al.* [6] simulated numerically NO_x emission of 800MW ultra-critical concurrent boiler with an intermediate reheat, in which the rate of over-fire air ranges between 15% and 25%. They pointed out that NO_x is from the initial state of combustion, in which there is reducing zone of NO_x. Critical factors that affect NO_x reaction rate in primary combustion zone and burnout zone are the O₂ concentration and the burning rate of coke. As the burnout air rises, the NO_x generation rate decreases in primary combustion zone. When the NO_x reduction zone enlarges, the NO_x emission reduces noticeably. The air-staged combustion processing has been applied to 2093 ton/h boiler in Jingjie power plant in Yulin, Shaanxi, China [7]. In order to reduce NO_x emission, the low NO_x nozzles are adopted, and over-fire air nozzles are increased. Over-fire air rate reaches 25%. Compared with 350-465 mg/m³ of NO_x emission before the reformation, it decreases by 60-70%. Besides, the temperature deviation of fuel gas decreases at the outlet of the furnace. Xiao *et al.* [8] investigated theoretically and experimentally the effects of air-staged combustion on temperature performance in 600 MW sub-critical boiler. It is found that air-staged combustion makes the temperature at the bottom of the screen increased by 80 °C and the temperature of super-heater increased by 20 °C without regulating operation plan of boiler. By comparing two 600 MW units with the air-staged combustion and low NO_x burner, Chen

et al. [9] pointed out that the air-staged technology could easily result in the lower boiler combustion efficiency, serious corrosion and slag formation. Zhou *et al.* [10] applied the air-staged technology to SG-420/13.7-W576 boiler. On the basis of experimental results, both the safety and economy can be improved with several techniques, such as: the reasonable air distributions to improve the burnout air ratio, the low oxygen combustion and the low NO_x burner.

On the basis of above research findings, NO_x generation rate is affected by fuel properties and combustion conditions when the air-staged technology is used. Fuel properties include fuel types, moisture, ash contents, and existing state of Nitrogen and coal fineness. Combustion conditions involve the ratio of fuel to air, the residence time of fuel gas after burning, the temperature in primary combustion region, the excess air coefficients, etc. These factors interact with each other. It is necessary to consider the above factors comprehensively so as to achieve an ideal effect of NO_x reduction.

At the present work, NO_x reduction is simulated numerically to obtain three dimensions of the flow fields, the temperature fields and the concentration fields during the air-staged combustion in 350MW units, the calculating results are analyzed.

2. Physical Model and Computation Method

Figure 1 shows a physical model of boiler. It's the model of the 1150 ton/h single drum reheat cycle boiler in Dalian Huaneng power plant in China. It's 14 m long, 12 m wide and 47 m tall. 13 layers of burners are installed at the four corners, each layer has four nozzles. The whole set of burner is composed of five layers of primary air nozzles, five layers of secondary air nozzles, one layer of oil nozzle and two layers of three-air nozzles. Sizes of each burner are shown in Figure 2.

In establishing the model, the four corners are removed in order to make the center of the furnace form four-corner tangential firing. Nozzles of the burners are simplified, and the effects of both the surface of heat transfer and other devices are neglected.

Physical model of the boiler is grid with Gambit included in fluent software. The 3.5 million of hexahedral unstructured grid were adopted. The 0.3 was given for an interval size in the spacing item. The numerical simulation is developed with fluent software. A Realizable two equations model is applied to calculating

turbulent flowing. A P1 model is used to consider the radiation heat transfer. A discrete phase model is employed to treat with pulverized coal particle transport and an Eddy-Dissipation is adopted for chemical reaction. High-volatile coal serves as a fuel, air works as a combustion adjutant, and the dynamic and diffusion model is adopted to simulate combustion. The simple solver with the velocity-pressure coupling is employed. Boundary conditions are given, including velocity inlet and pressure outlet. The wall temperature is set at 500 K, the temperature of primary air at 350 K, and temperature of secondary air at 570 K. The convergence criterion is set as 0.001 for the momentum equation, the continuity equation and species mass fraction equations. However, the convergence criterion is defined as 1×10^{-6} for the energy equation and P1 radiation equations.

The formation of NO_x involves the thermal and fuel NO_x, but little prompt NO_x. Here, only the production of NO was taken into account because NO_x emitted into the atmosphere from the burning fuels consists mostly of NO, with much lower concentrations of NO₂ and N₂O. The concentration of thermal NO_x is calculated with the extended Zeldovich mechanism, described in Equations (1-3).

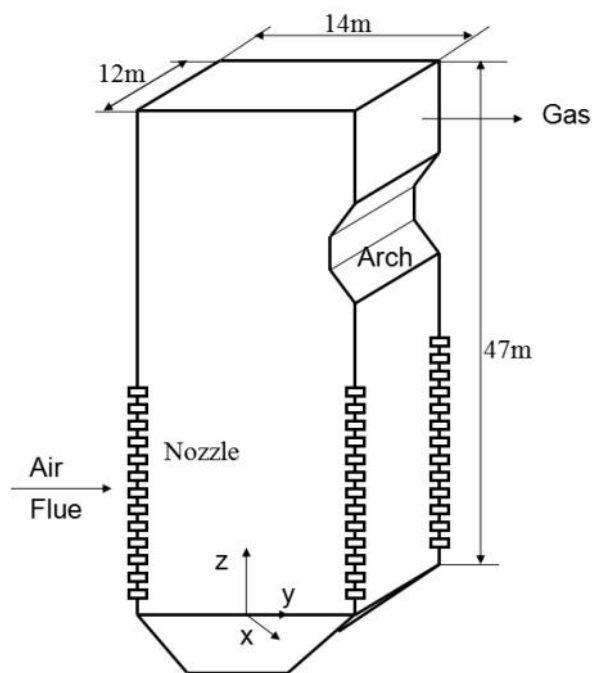
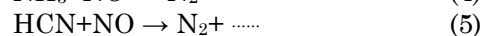
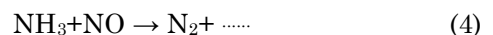


Figure 1. A physical model of boiler

The fuel NO_x contributes mostly to the formation of NO about more than 75%. The nitrogen content in the fuel is generally composed of volatile-nitrogen (HCN and NH₃). It is found that HCN is the center of the whole reaction loop. Both NH₃ and HCN originate from the Fuel-N, and NH₃ can be converted into HCN provided that large quantity of NO exists in the fuel-rich environment, while HCN can't be turned into NH₃ directly. In the lean-oxygen atmosphere, HCN and NH₃ can be reduced into N₂ as described in Equations (4-5). The formation of prompt NO_x was neglected in the calculations.



3. Results and Discussion

NO_x emission is simulated by a computation fluid dynamics software FLUENT 6.3.26 in both air-staged and non air-staged boiler combustion. In the non air-staged combustion, two layers of three-air nozzles are removed, the velocity of secondary air is 46 m/s and the mass flux of pulverized coal is 144 ton/h. In the air-staged combustion, two layers of three-air nozzles remain and the velocity of secondary air is 35 m/s. Effects of air-staged combustion on NO_x reduction at 20% of burnout air are analyzed by comparing computation parameters including temperature fields, flow fields and

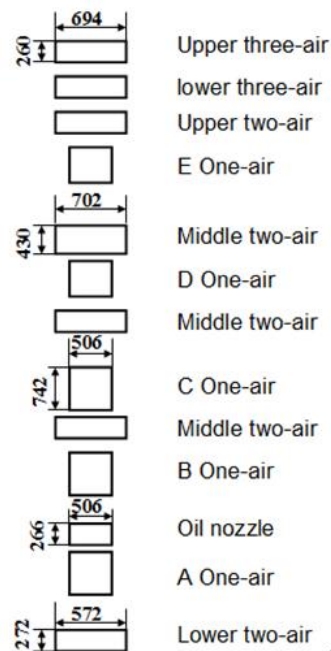


Figure 2. Nozzle location and size of burners

NO_x concentration fields.

3.1. Temperature fields

Figure 3 shows temperature distributions on the transverse cross-section in the primary combustion region. This cross-section locates in the center of the whole set of burners, i.e. the outlet of third-layer nozzles; its height is 6.184 m. It embodies the temperature performance in the primary combustion region.

As can be seen from Figure 3, the temperature along the direction of pulverized coal jet is higher in primary combustion region. This is caused by the released heat of pulverized coal ejection due to its quick ignition. However, the temperature is lower at the surroundings of the primary burners and on the furnace wall. These characteristics are attributed to the structure of the four-corner tangentially pulverized coal boiler, which is beneficial to the ignition and combustion of pulverized coal.

When the temperatures in the air-staged and non air-staged combustion are compared, it is found that higher temperature appears in the center of primary combustion region in the former while it occurs in the vicinity of burner ejector in the latter. Furthermore, when air is staged, both lower temperature deviation of fuel gas and uniform temperature can be obtained.

Figure 4 shows the temperature distributions on the longitudinal cross-section in the center of the furnace, which clearly gives an overview of temperature distributions. The temperature is higher in the primary combustion region than in other regions; the highest value appears in the vicinity of primary burner

ejector. When the air is not staged, the flame is easily attached to the furnace wall and the temperature deviation of fuel gas is higher in the furnace. However, when the air is staged, the flame concentrates in the center of the furnace forms a torch and showing lower temperature deviation of fuel gas.

3.2. Flow fields

Figure 5 shows the velocity distributions on the transverse cross-section in primary combustion zone. Fuel gas rotates clock-wisely on the transverse cross-section in primary combustion zone, forming proper rotating velocity field, which is conducive to organizing the combustion of pulverized coal. The higher velocity appears at the outlet of the burner, while the lower velocity occurs in the center of the furnace and around the wall. When the air is not staged, the fuel gas interacts with each other intensively due to the higher secondary air velocity, easily causing the flame to attach to the furnace wall. Besides, the streamline appears approximately a circular shape while it appears an ideal tangent circle in the center and vortexes around the wall of the furnace when the air is staged. This contributes to the ignition and combustion of pulverized coal and the NO_x reduction reaction.

Figure 6 shows velocity distributions on the longitudinal cross-section of the furnace. Streamline is smooth and has enough rigidity under the air-staged circumstances. However, it has poor rigidity and appears vortexes on two sides of primary combustion zone under no air-staged circumstances, which causes the flame to attach to the wall and results in higher temperature deviation of fuel gas. This is consistent

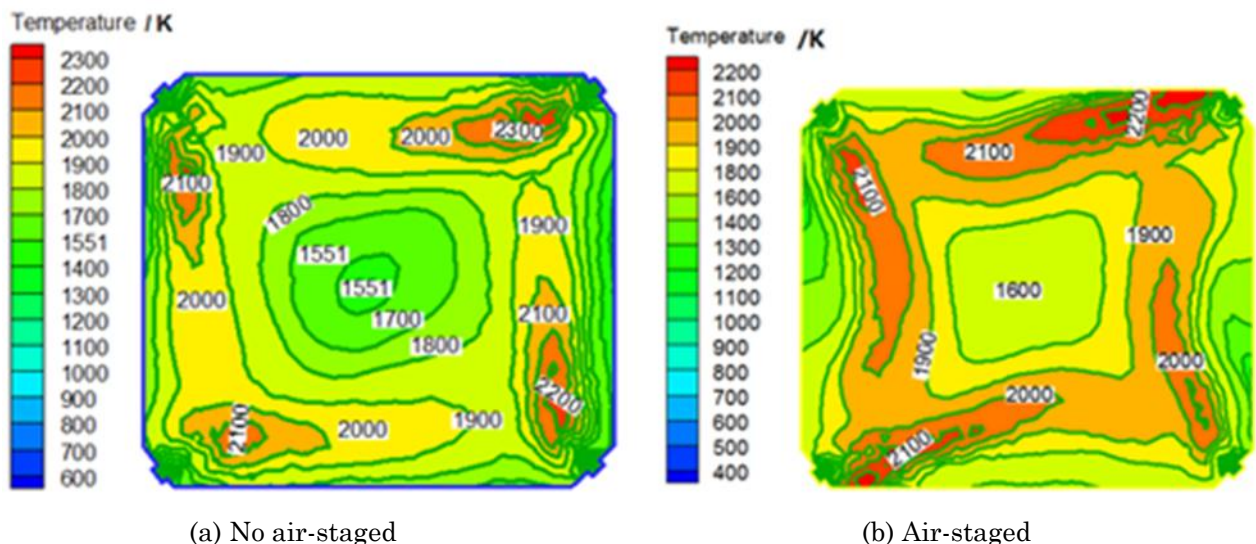


Figure 3. Temperature distributions on transverse cross-section in primary combustion region

with reference [8] which mentions that improving burnout air may achieve stronger rigidity streamline so as to ensure that the whole furnace is covered with flame and to regulate the deviation temperature of fuel gas.

3.3. NO mass fraction

Figure 7 shows NO mass fraction in the primary combustion zone for no air-staged and air-staged. NO mass fraction is higher near the outlet of the burner. This is mainly because the higher temperature and fuel concentration improve the thermal and fuel NO generation rate near the outlet of the burner. However, in the region far from the nozzle of the burner, lower O₂ concentration contributes to the NO reduction.

Figure 8 shows the NO mass fraction on the longitudinal cross-section in primary combustion zone. NO mass fraction is higher near the outlet of the burner; it reduces along the height of the furnace. On the longitudinal cross-section, NO mass fraction is higher obviously for no air-staged than for air-staged.

3.4. Temperature distributions and NO mass fraction at the furnace outlet

Figure 9 shows a comparison of temperature distribution at the furnace outlet under the air-staged and the no air-staged. Compared to no

air-staged, the deviation of gas temperature at the furnace outlet is smaller for the air-staged, the zone of local high temperature is narrower, the temperature distribution is more uniform, and the heat exchanger arranged at the exit of furnace will obtain more stable temperature characteristics. This is mainly because the air is given into the furnace by means of step-by-step, which makes the mixture of both the fuel and the air more uniform, the fuel burns more fully. Therefore, there are few area of local high temperature, so the stable temperature characteristics can be achieved.

Figure 10 shows NO mass fraction at the furnace outlet. Compared to the no air-staged, NO mass fraction at the furnace outlet is lower under the air-staged. This is mainly because both NH₃ and HCN in the gas reduced obviously for the air-staged, which is caused by the fuel without sufficient oxygen. In addition, the ions of both the O and N in the gas under the air-staged are significantly decreased, which is the result of the air-staged to reduce the local high temperature of the flue gas. Therefore, the application of air-staged combustion can effectively reduce the concentration of NO emissions, improve the air quality, and promote the environmental protection.

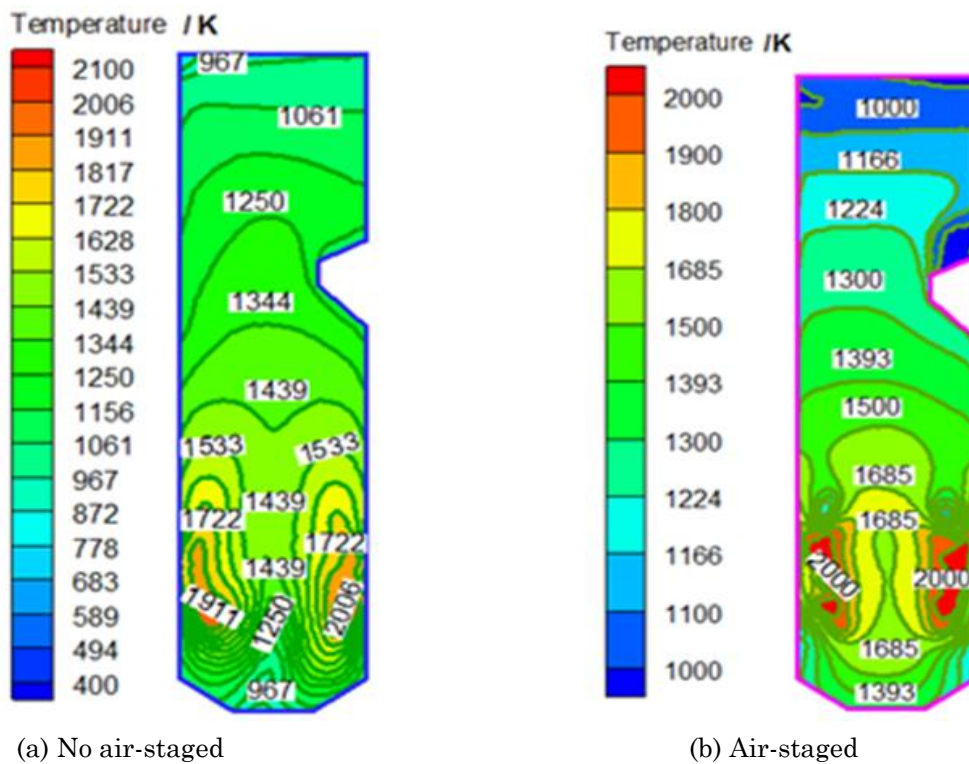
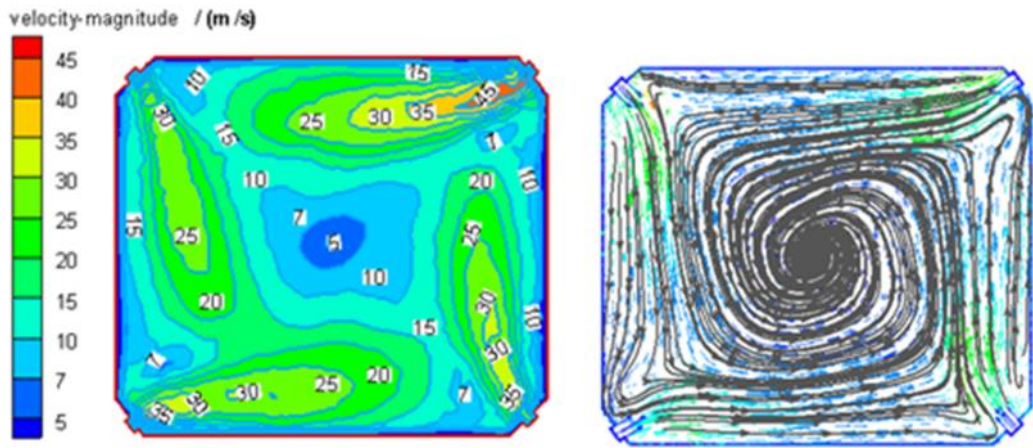
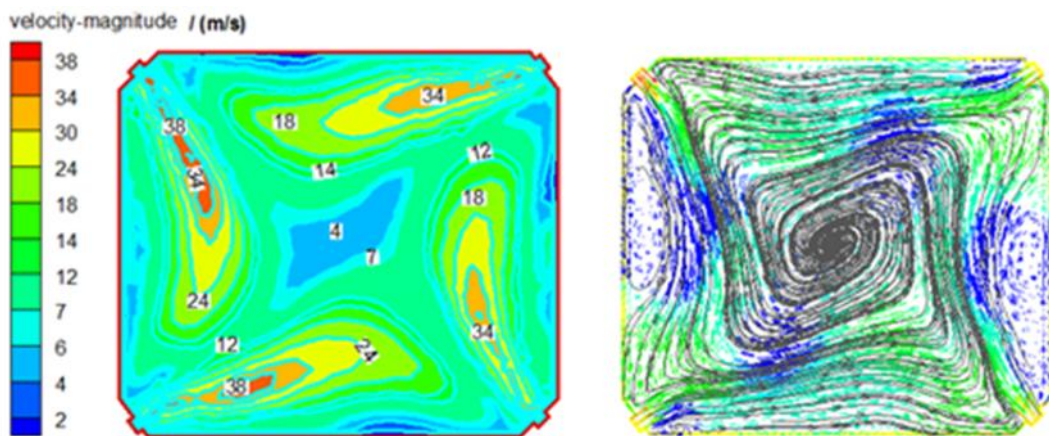


Figure 4. Temperature distributions of longitudinal cross-section in the center of the furnace

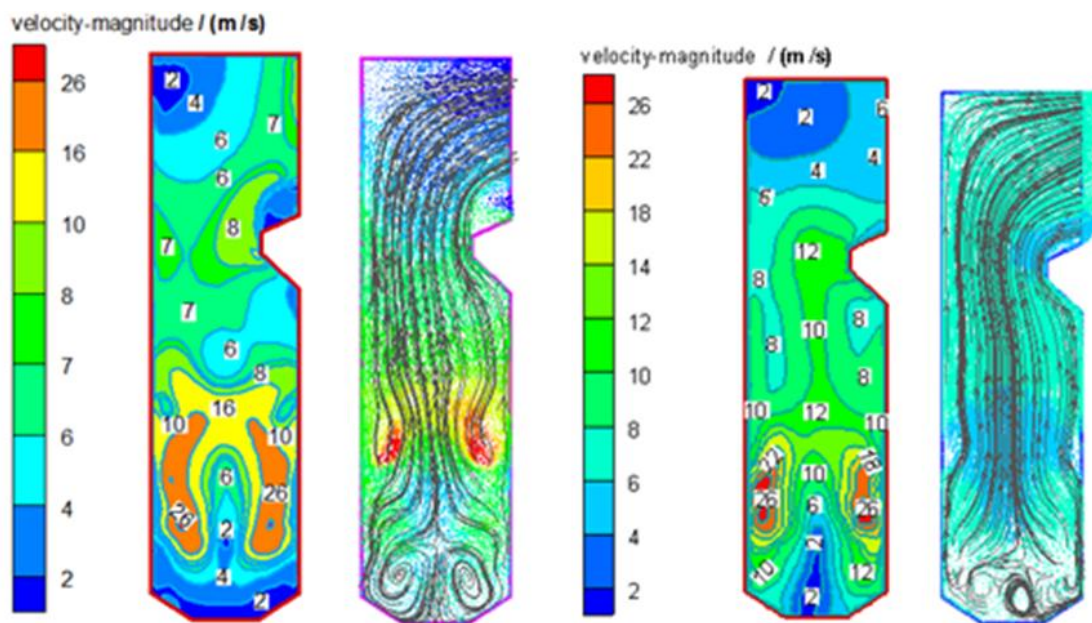


(a) No air-staged



(b) Air-staged

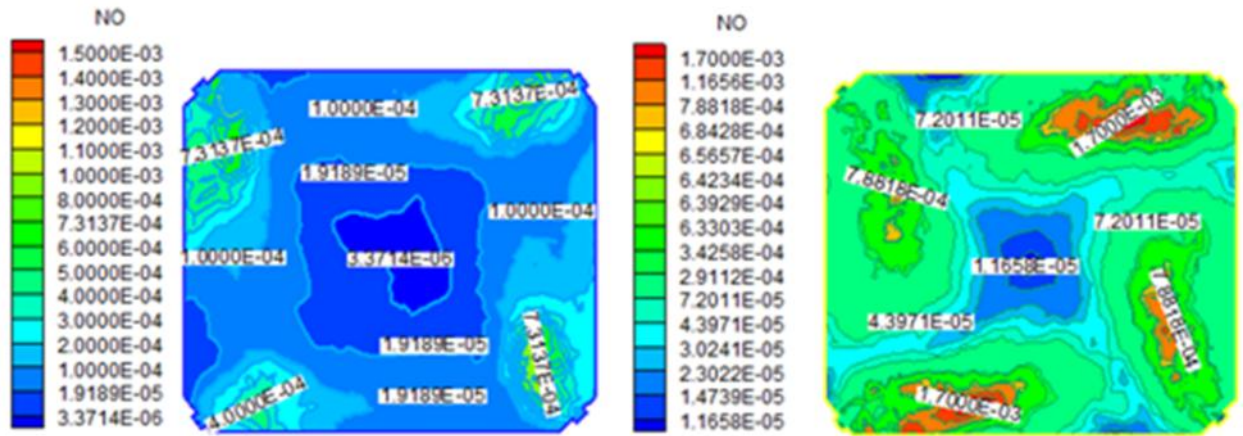
Figure 5. Velocity distributions on transverse cross-section in primary combustion zone



(a) No air-staged

(b) Air-staged

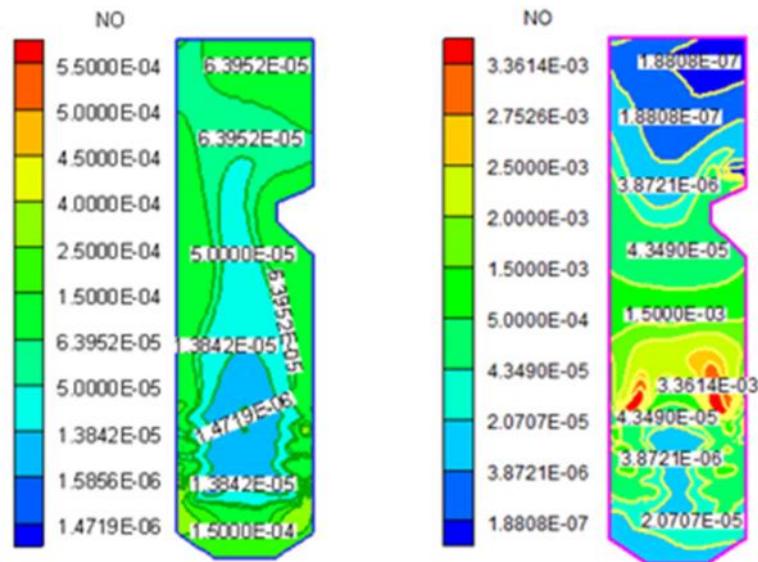
Figure 6. Velocity distributions at longitudinal cross-section of the furnace



(a) No air-staged

(b) Air-staged

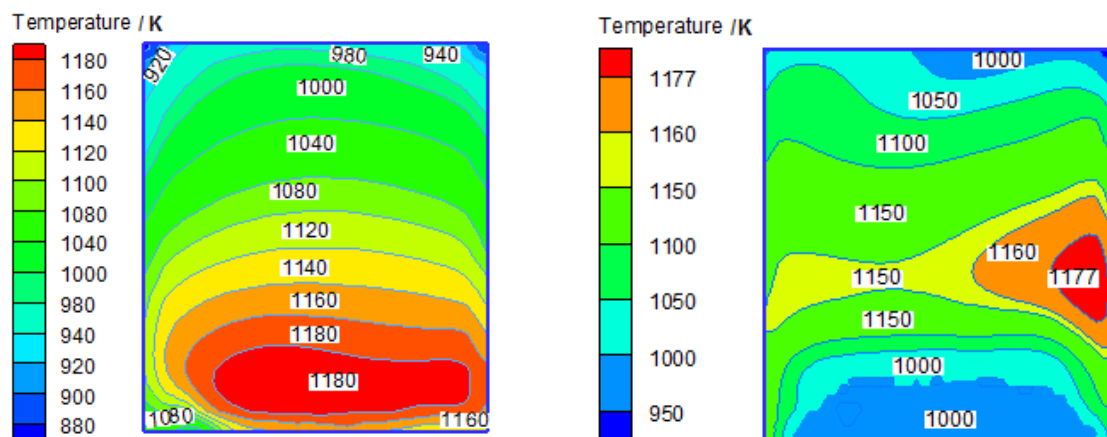
Figure 7. Velocity distributions on transverse cross-section in primary combustion zone



(a) No air-staged

(b) Air-staged

Figure 8. NO mass fractions on the longitudinal cross-section



(a) No air-staged

(b) Air-staged

Figure 9. Temperature distributions at the furnace outlet

3.5. A comparison of the present results to the calculated values in the reference

By comparing the calculated results in this article to that of the literature [11], it found that the temperature field distribution in this article is in agreement with that under the optimal operating conditions in literature [11]. The temperature in the region near the burner is higher, and the temperature rises firstly and then reduced along the central axis of the furnace. The difference is that the temperature in the vicinity of the burner nozzle is higher than that in the literature [11], the maximum temperature is about 2000 K, while the value in the literature is only about 1700 K. The temperature in the literature at the burnout zone and the upper part of the furnace is higher than the simulation results in this article.

The differences between the simulation results in this article and that in the literature

are caused by a reasons that is to burn different kinds of coal. In this paper, the high volatile coal in the Fluent software library is adopted. However, a kind of Northern Shanxi bituminous coal is applied in the literature. Because the high volatile coal powder is easy to be ignited as compared to Northern Shanxi bituminous coal, more heat is released quickly at the near of the burners in my paper so as to a higher temperature.

4. Conclusions

At the present work, NO_x emission under the circumstances of air-staged combustion in the boiler is simulated numerically with the Fluent software. Various parameters, including the velocity fields, the temperature fields and NO_x concentration fields, are obtained. When air-staged technology is used, flow field in the furnace have a stronger rigidity and a

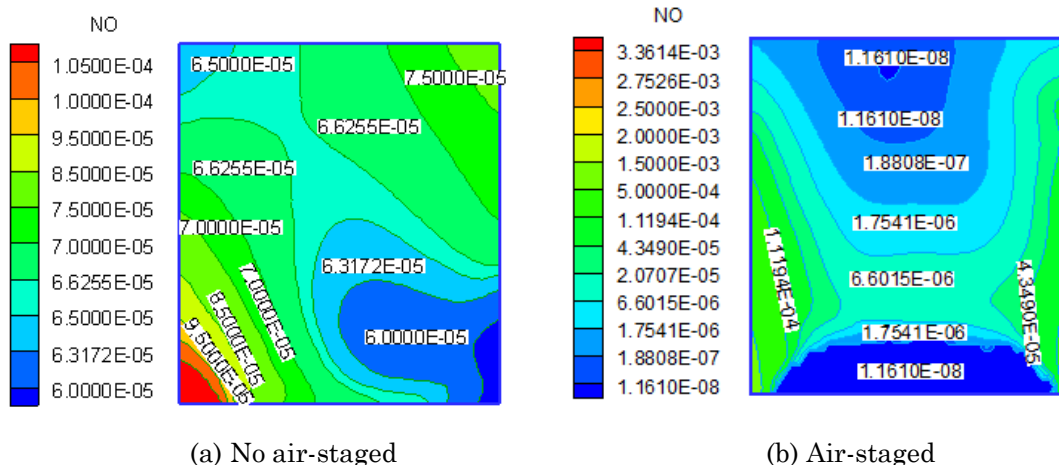


Figure 10. NO mass fraction at the furnace outlet

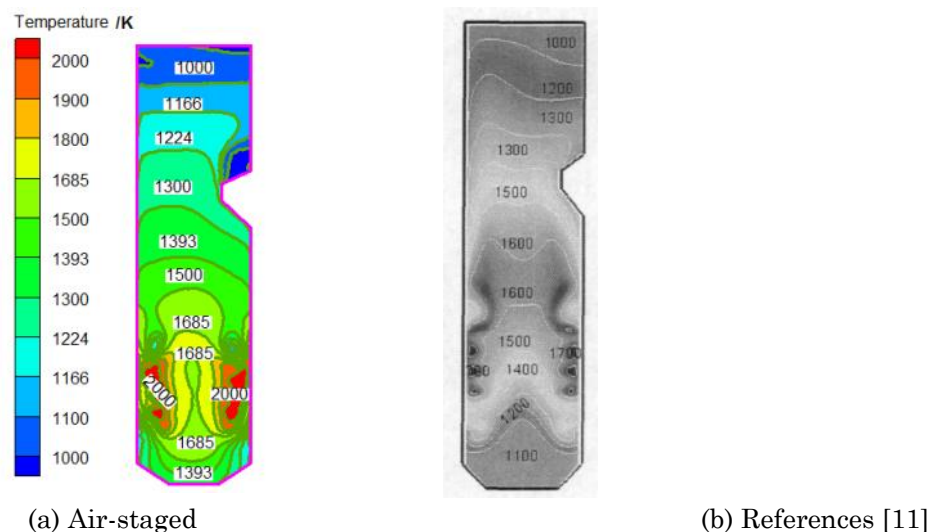


Figure 11. A comparison of the present result with the calculated results in the reference [11]

force of penetration. The flame is not easily declining and attached to the wall. Temperature distributes uniformly in the furnace, with the lower temperature deviation of fuel gas. Flame may form torch in the center of the furnace, with excellent performance of both convective and radiation heat transfer. NO mass fraction reduces along the direction of the height of the furnace, with lower value at the outlet of the furnace.

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